



Lesson 8



Thermomechanical Measurements for Energy Systems (MENR)

Measurements for Mechanical Systems and Production (MMER)

Measurement of Mechanical STRAIN

Strain measurements are perhaps the most widespread measurements done

in engineering (*tension, force, pressure, ...*):

$$\varepsilon_a = \frac{L_2 - L_1}{L_1} = \frac{\Delta L}{L}$$

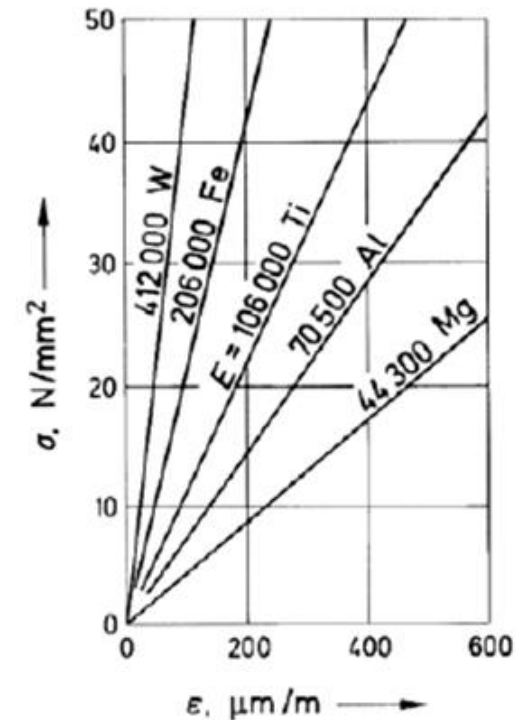
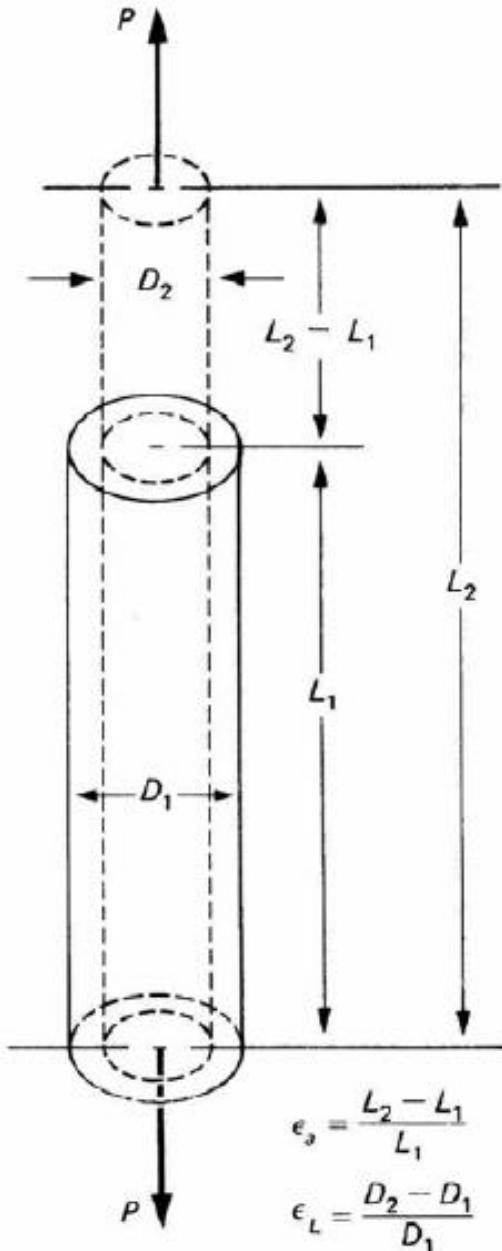
We measure here the *lengthening and shortening* of mechanical structures. If we know the **Young's Modulus E** of the material, after measuring the strain ε_a we can obtain the value of **mechanical tension $\sigma = E \cdot \varepsilon$** !

Transverse strain is measured by considering the *radial contraction* $(D_2 - D_1)/D_1$ and involves the knowledge of the

Poisson Coefficient: $\nu = \frac{\varepsilon_t}{\varepsilon_a}$

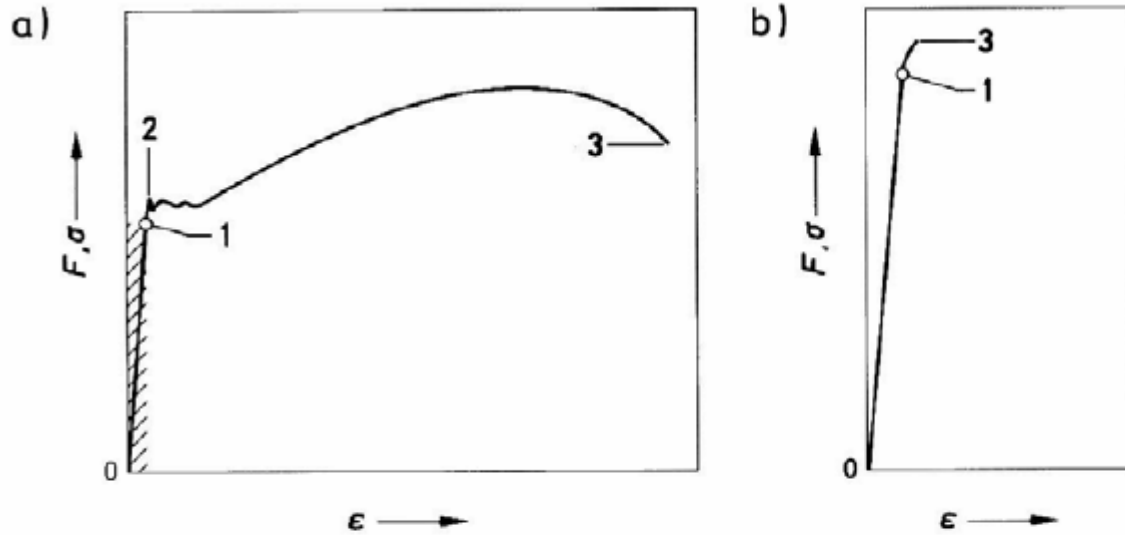
Which for metallic materials ranges from 2 up to 4 ...

Conventionally **$\nu = 0,3$**



W	tungsten
Fe	iron (steel)
Ti	titanium
Al	aluminum
Mg	magnesium

Ductile and brittle materials:



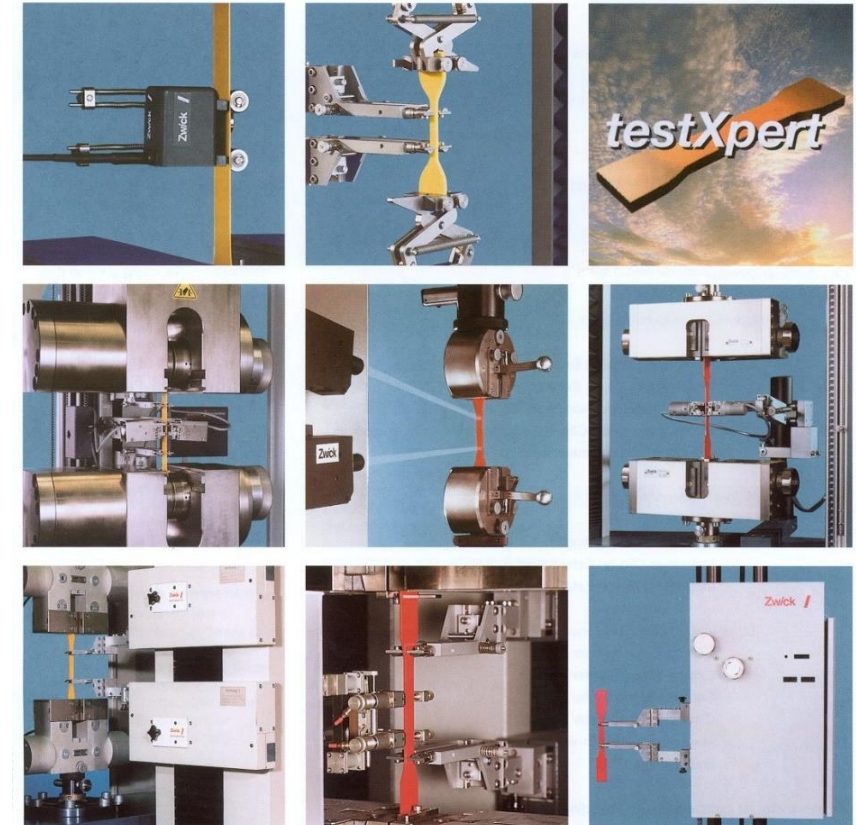
1. limit of elasticity,
2. yield point,
3. fracture.

Examples of force/strain and stress/strain curves.

a) Characteristic of a ductile material with large yield range

b) Characteristic of a brittle material

A first type of instruments used to measure the *strain* or the *displacement* between two adjacent points are the **Mechanical Extensometers ...**



Mechanical Extensometers :

The displacement is measured between two points, initially at distance l_0

When the specimen elongates Δl , the indicator rotates of an angle ϑ and we read the output λ ...

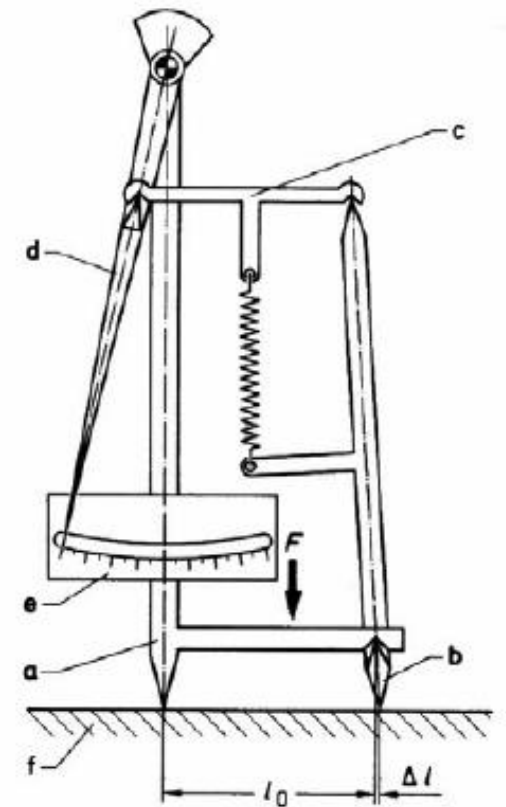
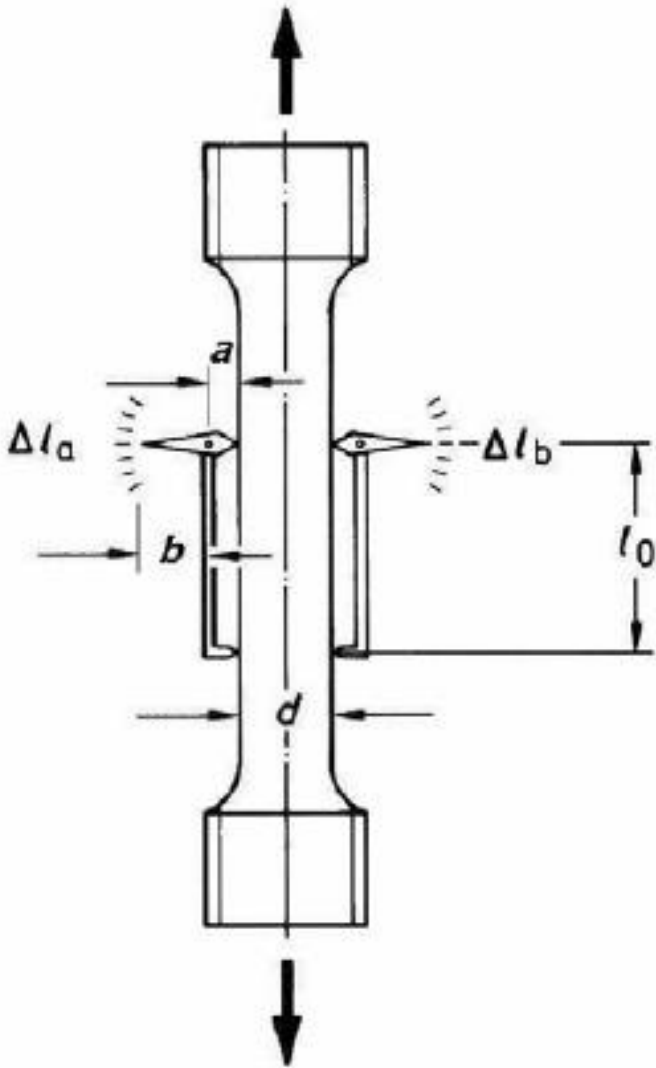
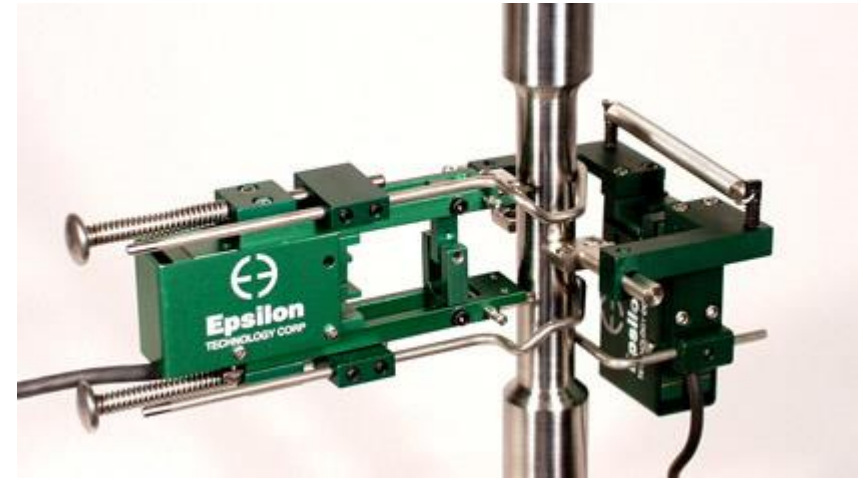
$$\Delta l = a \cdot \theta$$

$$\lambda = b \cdot \theta$$

The ratio is :
$$\frac{\Delta l}{\lambda} = \frac{a}{b}$$

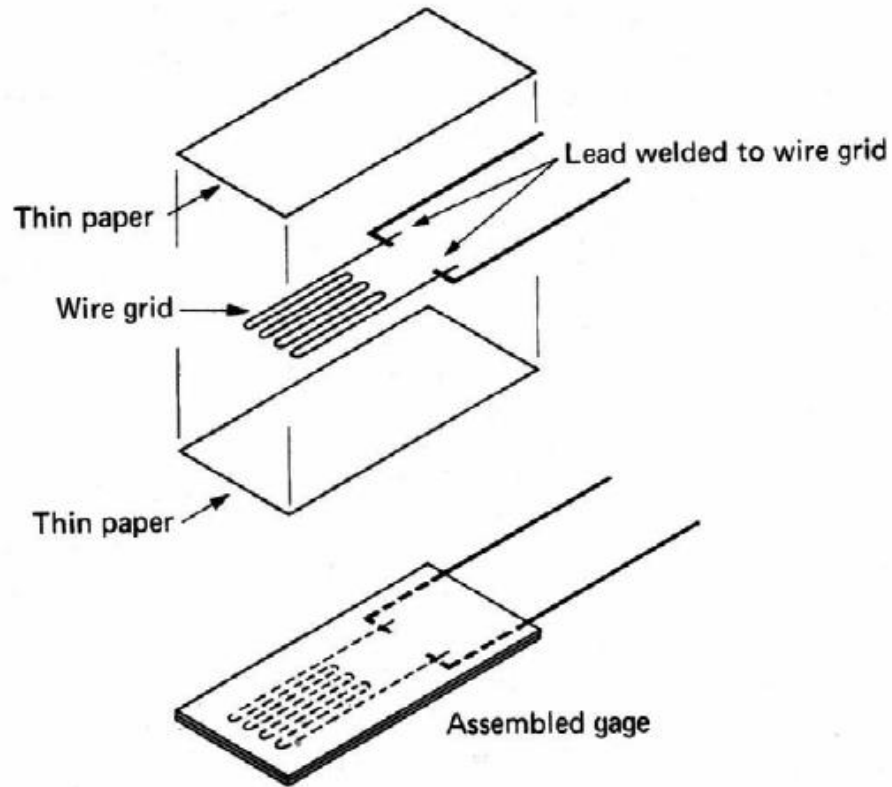
And the graduation curve is then:

$$\varepsilon = \frac{\Delta l}{l_0} = \frac{a\lambda}{l_0 b}$$



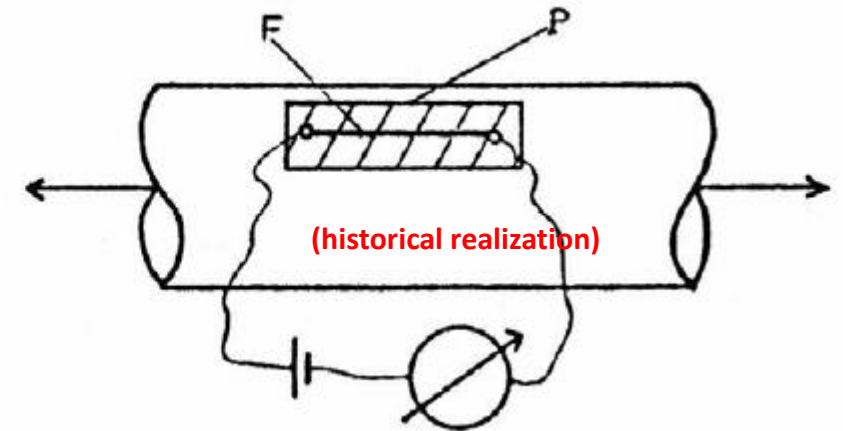
Electrical STRAIN GAGES :

$$\frac{\Delta l}{l} \rightarrow \text{Electrical Strain Gage} \rightarrow \frac{\Delta R}{R}$$



(modern realization)

P = parte del pezzo isolata
 F = filo di misura



However, what we really need is a *punctual strain measurement*, so to effectively monitor the zones of the surface where the strain concentrates.

Electrical strain gages are based on resistance variation with elongation !

$$R = \rho \frac{l}{S}$$

First we have to elaborate a bit the basic physical relation of **resistivity** ...

$$\ln R = \ln \left(\rho \frac{l}{S} \right) = \ln \rho + \ln l - \ln S$$

then we derivate the relation and we transform the *infinitesimal terms* into **finite differences** ...

$$\frac{1}{R} \frac{dR}{dt} = \frac{1}{\rho} \frac{d\rho}{dt} + \frac{1}{l} \frac{dl}{dt} - \frac{1}{S} \frac{dS}{dt}$$

$$\frac{\Delta R}{R} = \frac{\Delta \rho}{\rho} + \frac{\Delta l}{l} - \frac{\Delta S}{S}$$

Because it is $S = \pi \frac{D^2}{4}$ and $\frac{dS}{dt} = \frac{\pi}{4} 2D \cdot \frac{dD}{dt} \rightarrow \frac{\Delta S}{\Delta t} = \frac{\pi}{2} D \cdot \frac{\Delta D}{\Delta t} \rightarrow \Delta S = \frac{\pi}{2} D \cdot \Delta D$

$$\frac{\Delta S}{S} = \frac{\frac{\pi}{2} \cdot D \cdot \Delta D}{\frac{\pi}{4} \cdot D^2} = 2 \frac{\Delta D}{D} = 2 \cdot \varepsilon_t$$

it remains $\frac{\Delta R}{R} = \frac{\Delta \rho}{\rho} + \frac{\Delta l}{l} - 2 \frac{\Delta D}{D} = \frac{\Delta \rho}{\rho} + \varepsilon_a - 2\varepsilon_t = \frac{\Delta \rho}{\rho} + \varepsilon_a + 2 \cdot \nu \varepsilon_a$ (ε_t sign is the opposite of ε_a)

$$F = \frac{\frac{\Delta R}{R}}{\frac{\Delta l}{l}} = \frac{\frac{\Delta \rho}{\rho}}{\frac{\Delta l}{l}} + 1 + 2\nu$$

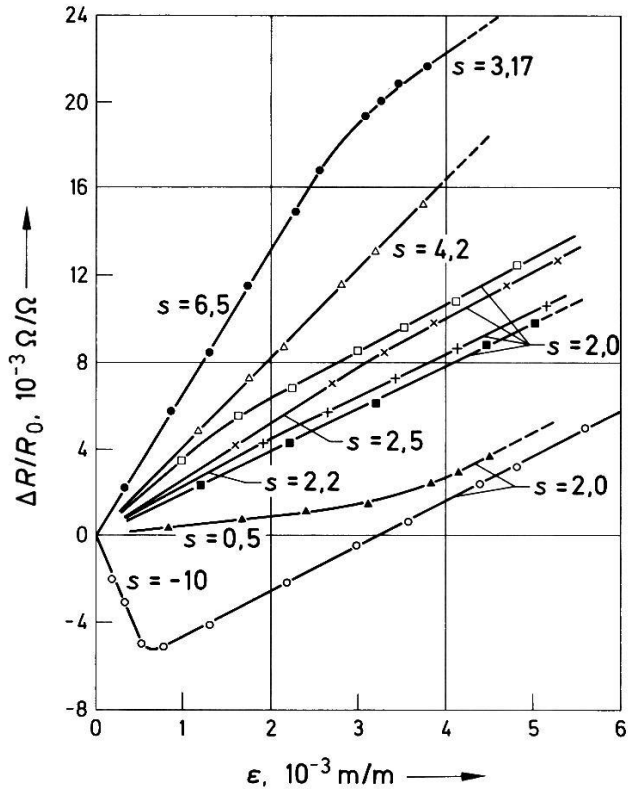
with $\nu \cong 0.3$ we would have $F \cong 1.5 \div 1.7$
however, variation of $\Delta \rho / \rho$ is not zero !

Because the contribution of the *resistivity variation* $\Delta\rho$ is difficult to calculate, strain gage producer *proceed experimentally* and «measure» the factor F for a certain number of gages.

The resulting number is then assigned to the same lot of transducer ... that is the **gage factor** $F \cong 2$

Strain gage «graduation curve»

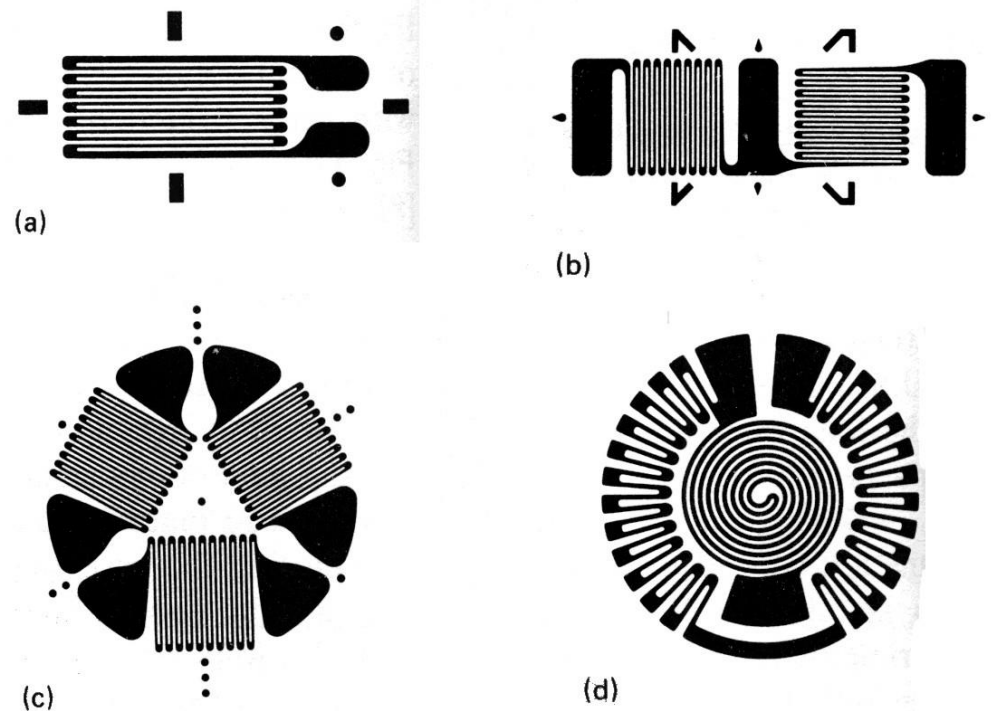
$$\frac{\Delta R}{R} = F \cdot \varepsilon$$



Note that $\Delta\rho/\rho$ is *almost constant* for strains up to 5-6 % !!

- Platinum-iridium 5/95
- △ Steel wire, spring steel (piano wire)
- "Eureka"
- x "Brightray C", hard
- + "Brightray C", annealed
- Soft iron
- ▲ "Manganin"
- Nickel "O"

Commercial strain gages :



Resistance/strain characteristics for freely tensioned wires

Thermal effects :

Unfortunately, because an electric current is flowing through every strain gage, there is an additional thermal

contribution on resistance variation: $\frac{\Delta R'}{R} = \alpha \cdot \Delta T$

Due to heating, the strain gage wire undergoes also a thermal elongation $\frac{\Delta l'}{l} = \beta' \cdot \Delta T$ which, moreover, is generally NOT equal to the thermal elongation of the underlying material $\frac{\Delta l''}{l} = \beta'' \cdot \Delta T$ We have therefore:

1. a resistance variation due to **mechanical strain**: $\frac{\Delta R}{R} = F \cdot \varepsilon$

2. a resistance variation due **only to thermal effects**: $\frac{\Delta R_T}{R} = [\alpha + F(\beta' - \beta'')] \cdot \Delta T$

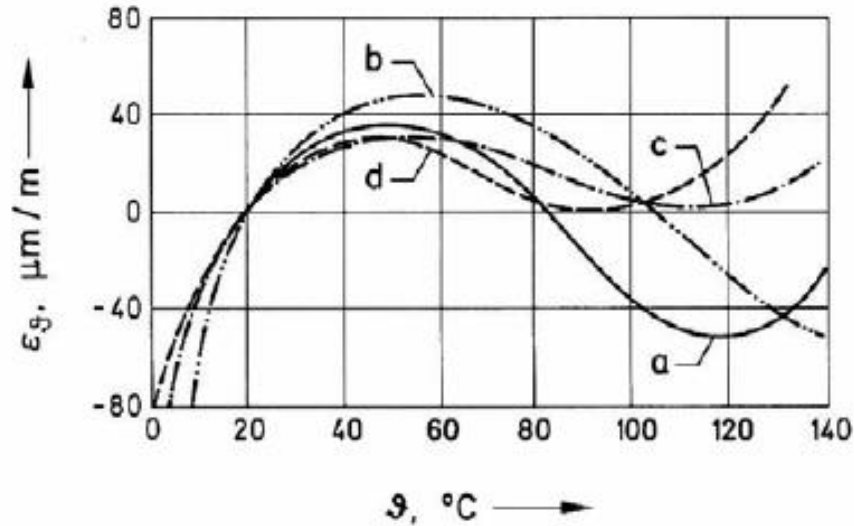
There is always an “**apparent strain**” which, in fact, is NOT real :

$$\varepsilon_{app} = \frac{1}{F} \frac{\Delta R_T}{R} = \left[\frac{\alpha}{F} + (\beta' - \beta'') \right] \cdot \Delta T$$

Because the coefficients α and β' depend on the gage wire material while the coefficient β'' depends on the underlying material, if we can make $\alpha + F(\beta' - \beta'') = 0$ then we are nulling the apparent thermal contribution ε_{app}

Temperature self-compensated strain gages approximately realize: $\frac{\alpha}{F} = \beta'' - \beta'$

Temperature self-compensation is not perfect, nor constant with temperature, and it works ONLY if the strain gage is applied on the “correct underlying material” ...

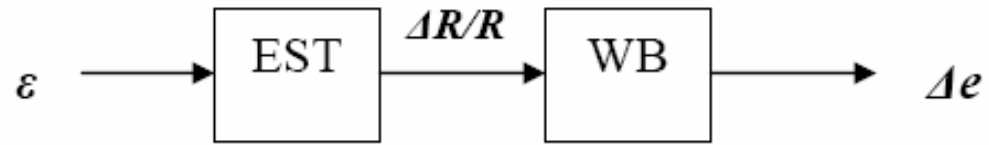


material	α_c
a titanium Ti6Al4V	$8.5 \cdot 10^{-6}/\text{K}$
b ferritic steel	$12 \cdot 10^{-6}/\text{K}$
c austenitic steel	$16 \cdot 10^{-6}/\text{K}$
d aluminum AlCuMg2	$23 \cdot 10^{-6}/\text{K}$

If $\epsilon = \frac{1}{F} \frac{\Delta R}{R}$ how much is actually the resistance variation $\Delta R = \epsilon \cdot F \cdot R$ if we measure a strain of

$100\mu\text{m}/\text{m}$ with a 120Ω gage ? ... $\Delta R = 100 \cdot 10^{-6} \times 2 \times 120\Omega = 0.024\Omega = 24\text{m}\Omega$

It's a variation of only the 0,02% of the base resistance ! These small variations are optimal to be measured with the **Wheatstone Bridge** ...



We have now a *combined measurement chain*:
 “Strain Gage + Wheatstone Bridge”

The two graduation curves work then together ...

$$\varepsilon = \frac{1}{F} \frac{\Delta R}{R} \qquad \frac{\Delta e}{E} = \frac{1}{4} \frac{\Delta R}{R}$$

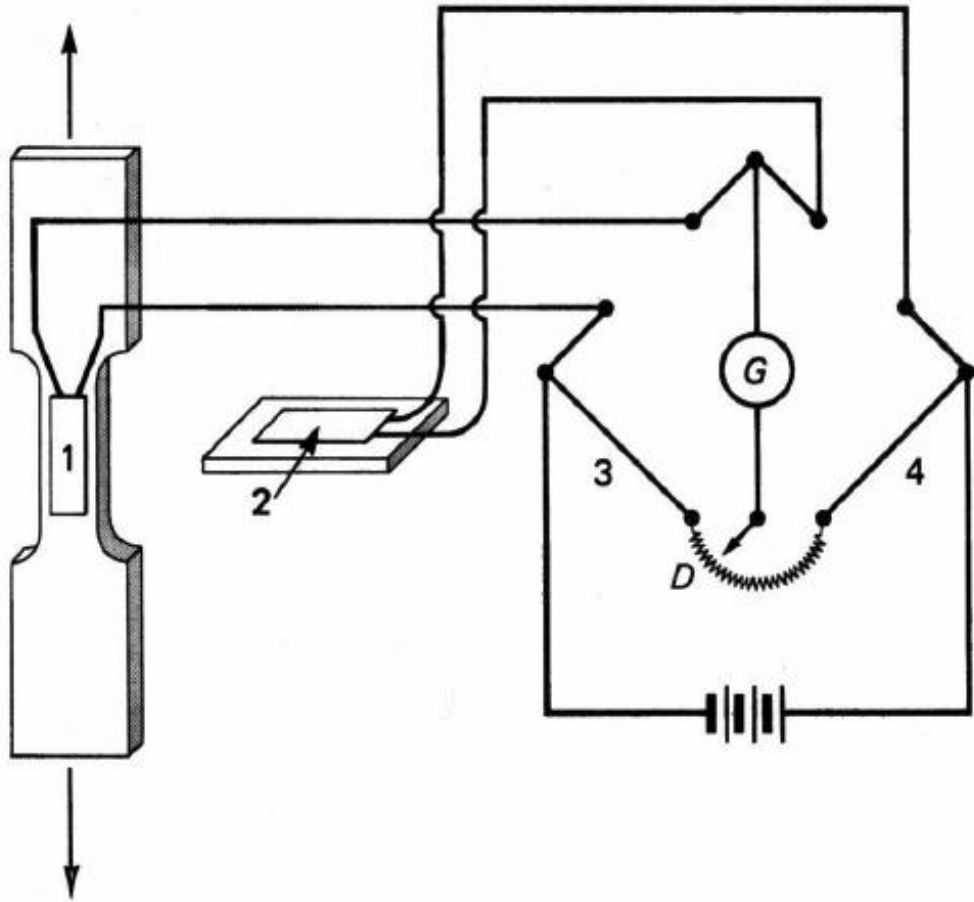
Which combined give us: $\frac{\Delta e}{E} = \frac{1}{4} \cdot F \varepsilon$

$$\Delta e = \frac{1}{4} EF \cdot \varepsilon$$

the combined graduation curve !

Strain gage (2) *does not measure any strain* but, it *undergoes the same thermal effects* of strain gage (1)

$$\frac{\Delta R_{T1}}{R} = \frac{\Delta R_{T2}}{R} = [\alpha + F(\beta' - \beta'')] \cdot \Delta T$$

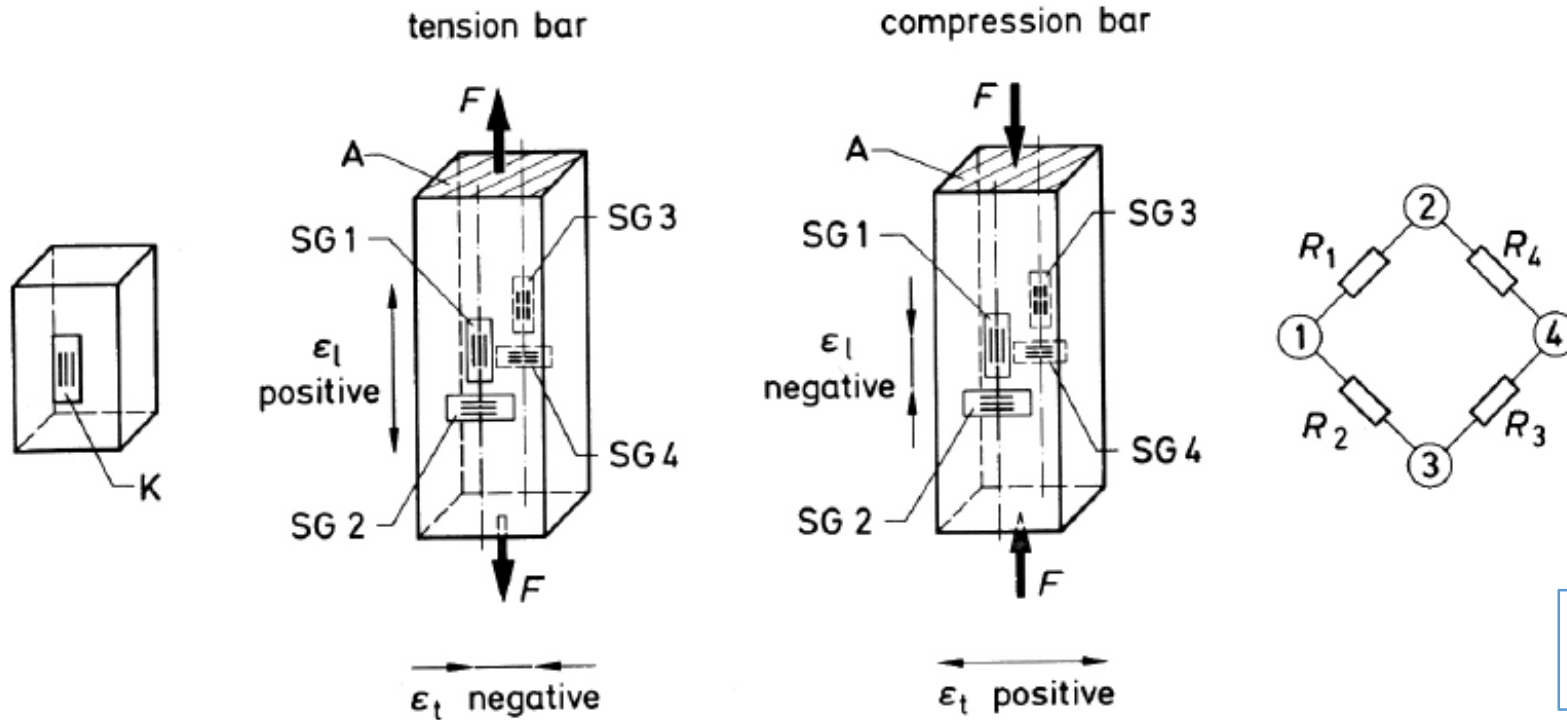


If the two strain gages (1) and (2) are applied on contiguous arms of the Wheatstone Bridge, the thermal effects

are “automatically eliminated”:
$$\frac{\Delta e_T}{E} = \frac{1}{4} \left(\frac{\Delta R_{T1}}{R} - \frac{\Delta R_{T2}}{R} \right) = 0$$

This is perhaps the *most important service* the WB does during strain measurement !

Measurement of tensile strain:

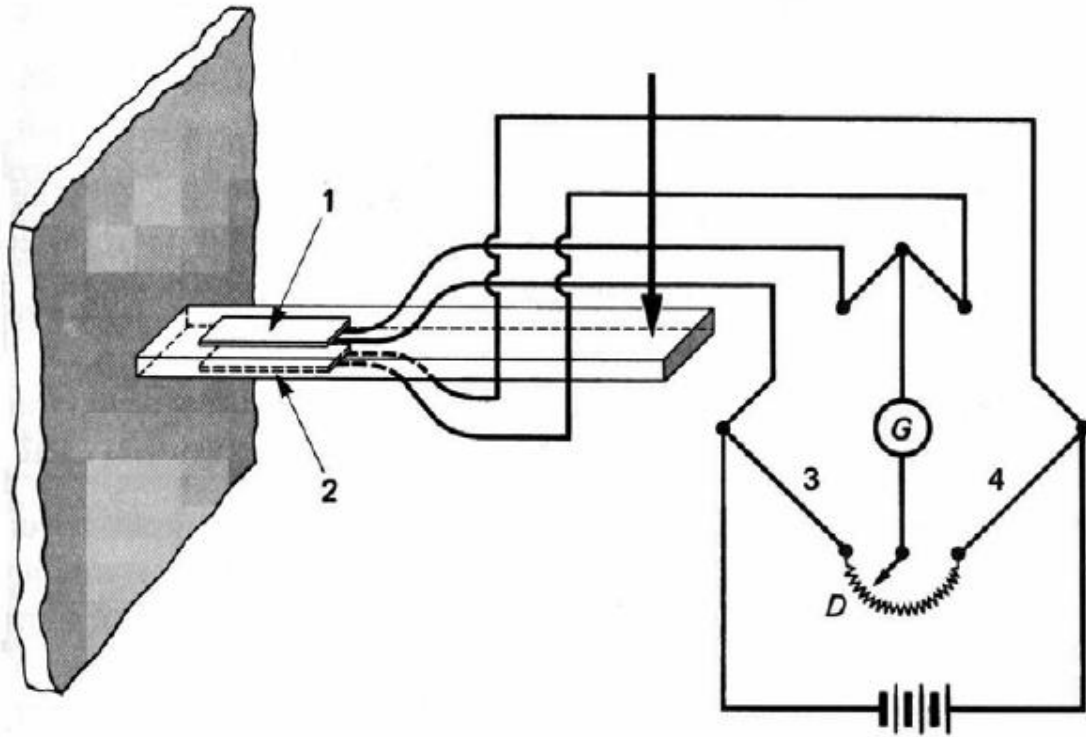


The second gage (2) can be applied more efficiently on the specimen to increase the measurement sensitivity, but it must be rotated 90° in a XY configuration to give a useful signal. It will then measure the transversal strain $\epsilon_t = -\nu \cdot \epsilon_a$ producing the graduation curve:

$$\frac{\Delta e}{E} = \frac{F}{4} (\epsilon_a - \epsilon_t) = \frac{F}{4} (\epsilon_a - (-\nu \epsilon_a)) = \frac{F}{4} \epsilon_a (1 + \nu)$$

where the factor $(1 + \nu) = 1,3$ is an extra amplification *due only to the bridge configuration*, called «**bridge factor**»

Measurement of bending strain:



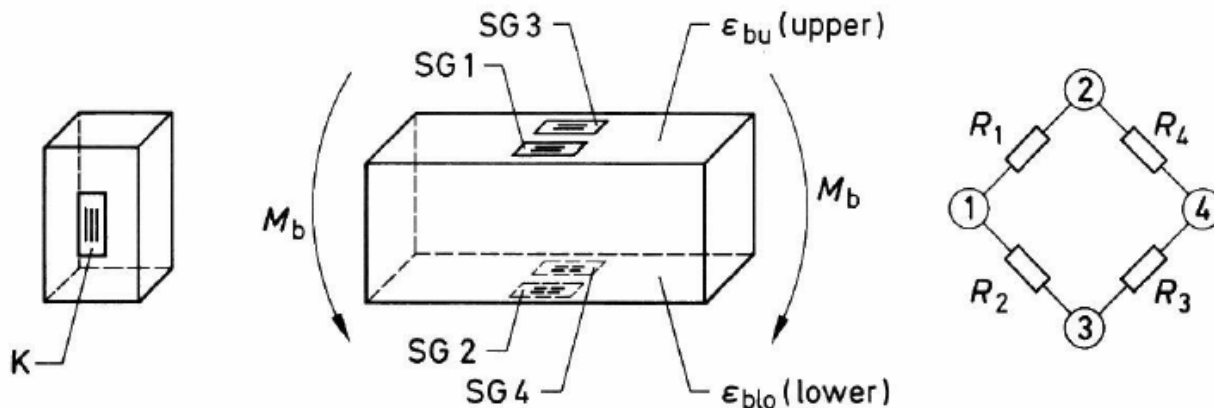
the upper gage measures the stretched fiber strain while the bottom gage measures the compressed fiber strain :

$$\frac{\Delta e}{E} = \frac{1}{4} F (\varepsilon_{f1} - \varepsilon_{f2})$$

$$\frac{\Delta e}{E} = \frac{F}{4} (\varepsilon_{f1} - (-\varepsilon_{f1})) = \frac{F}{4} \cdot 2\varepsilon_f = \frac{F}{2} \varepsilon_f$$

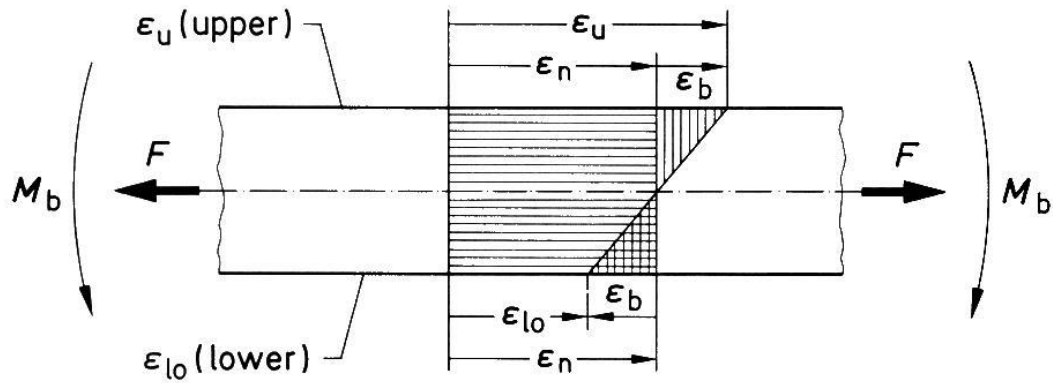
because it is $\varepsilon_{f2} = -\varepsilon_{f1}$

Ad the “bridge factor” is equal to 2 !



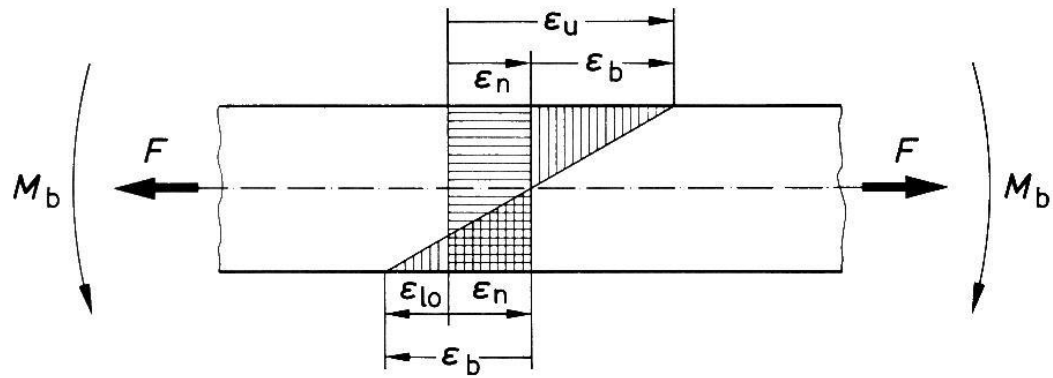
Note that the *bridge factor can be increased up to 4* if we employ four strain gages connected as shown on the left and make the full bridge active !

$$\varepsilon_1 = \varepsilon_3 = -\varepsilon_2 = -\varepsilon_4$$



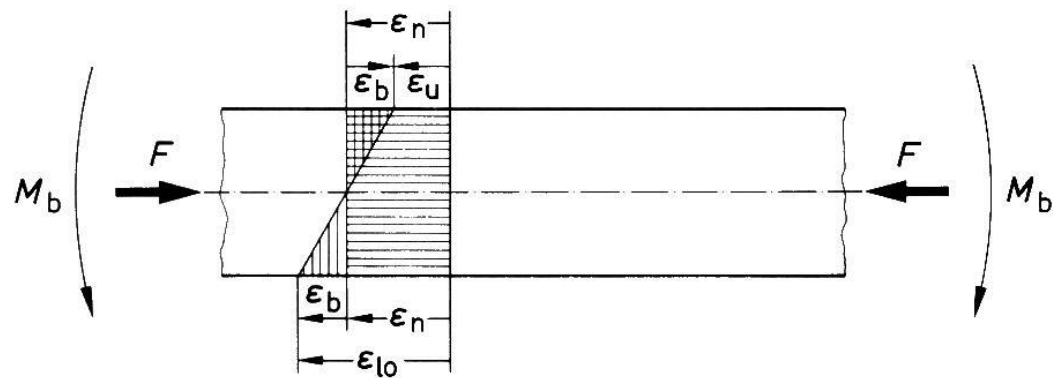
case a: $\epsilon_n > \epsilon_b$

ϵ_u and ϵ_{lo} are both positive



case b: $\epsilon_n < |\epsilon_b|$

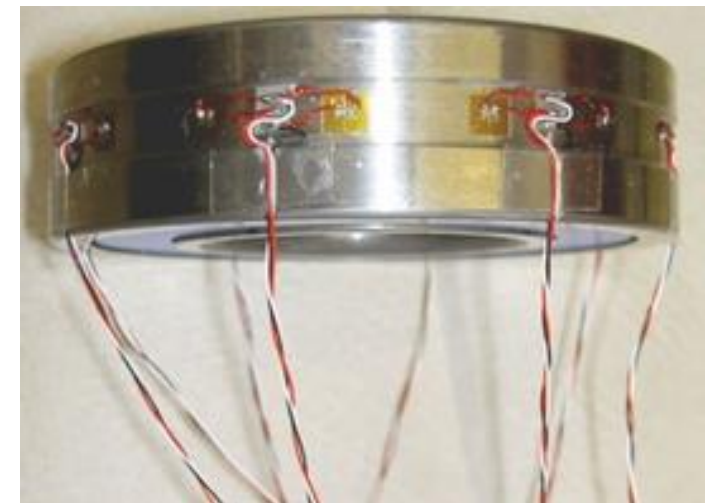
ϵ_u positive,
 ϵ_{lo} negative



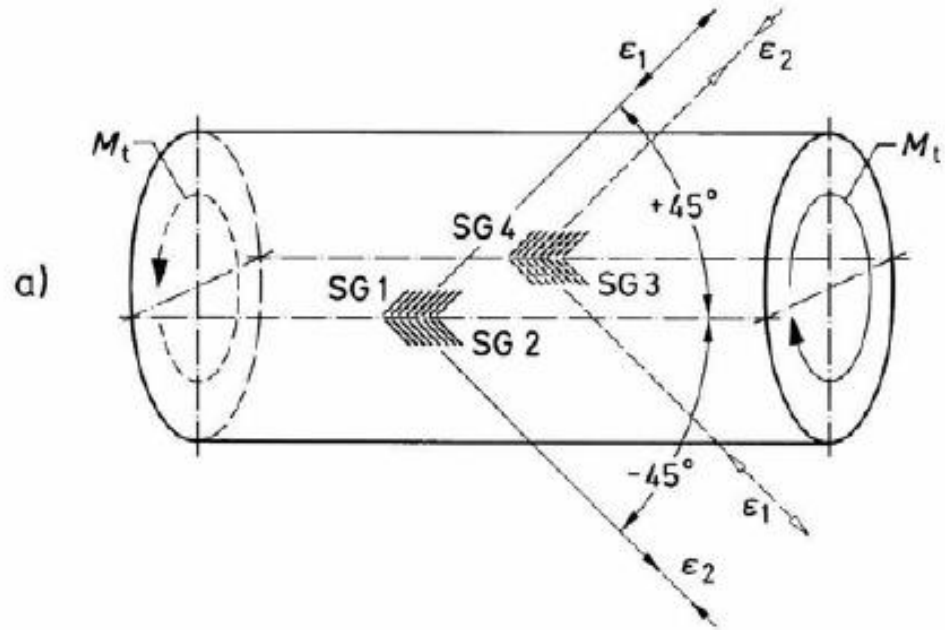
case c: $-\epsilon_n > \epsilon_b$

ϵ_u and ϵ_{lo} are both negative

Strains, of course, can be composed in more complicated ways, as shown in the figures ...



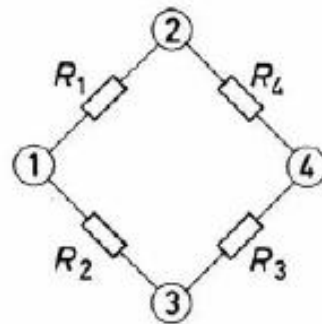
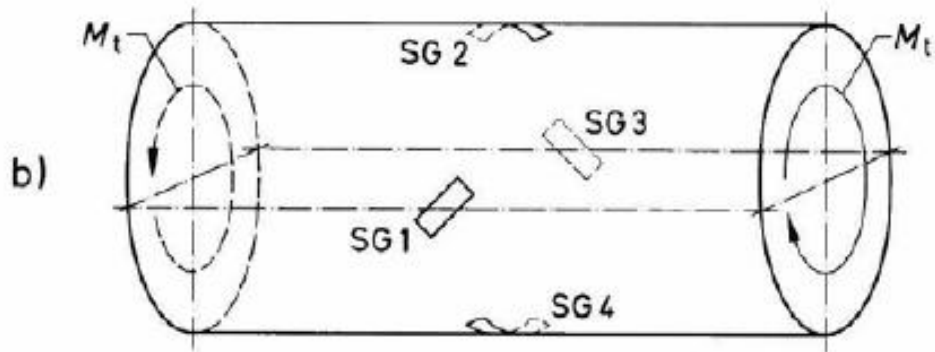
Measurement of torsion strain:



Fundamental application for the crankshafts, where we want to measure the **drive torque** to get the **power output** of an engine:

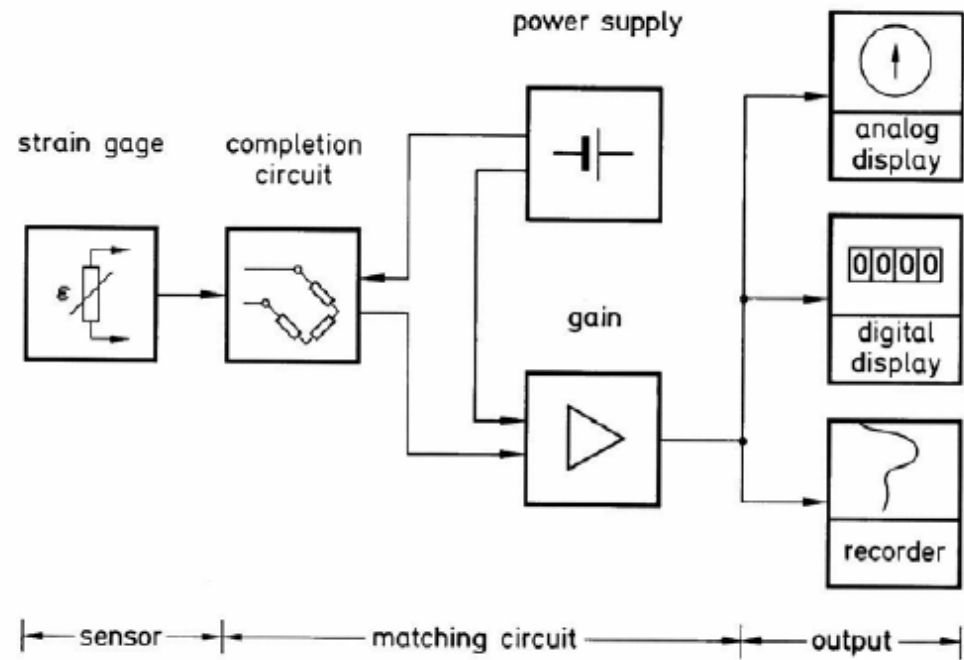
$$W_{mecc} = C \times \omega$$

Strain gages are placed as in the figure, aligned with the maximum strain 45° lines on the shaft surface. The strains will be $\epsilon_1 = \epsilon_3 = -\epsilon_2 = -\epsilon_4$ and the graduation curve results:

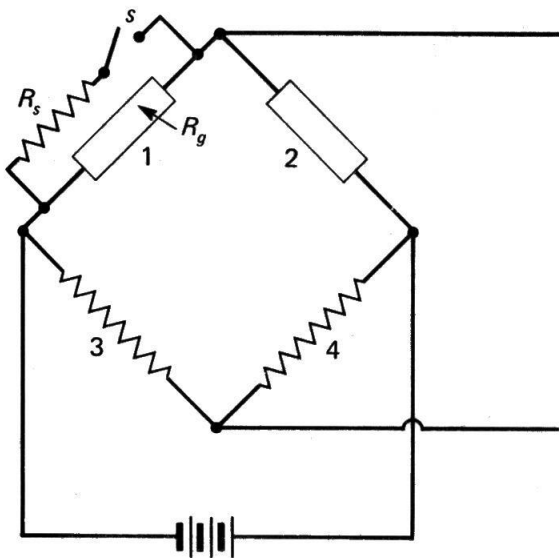
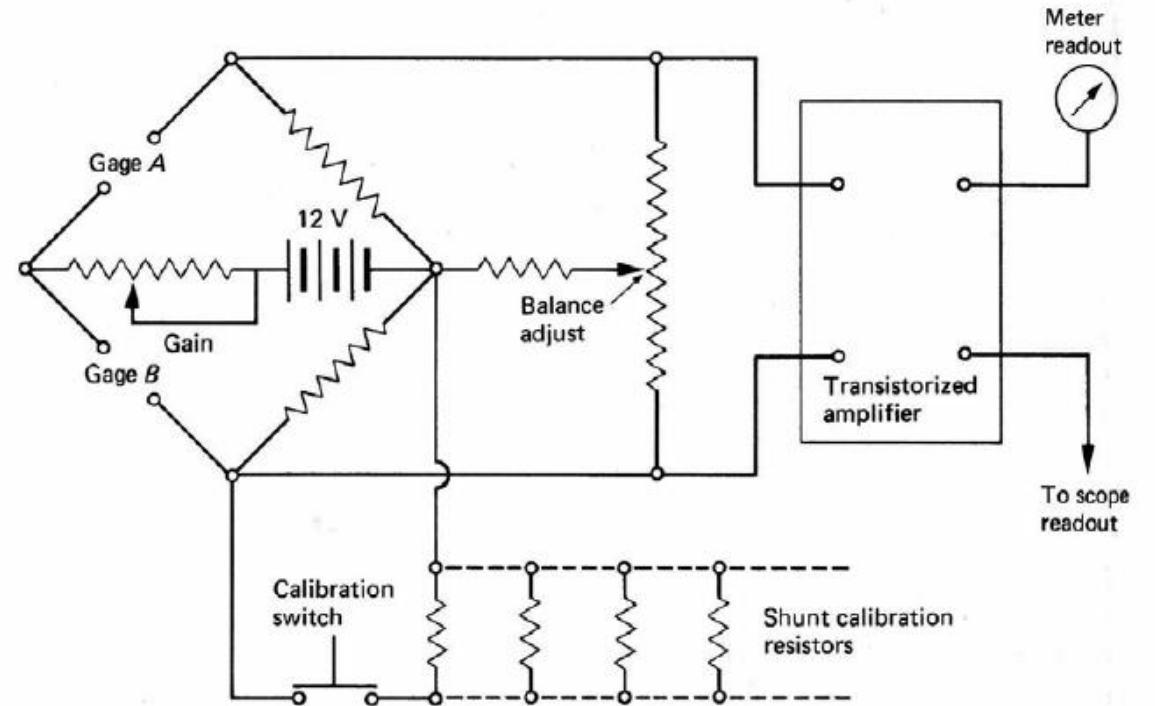


$$\frac{\Delta e}{E} = \frac{F}{4} (\epsilon_1 - \epsilon_2 + \epsilon_3 - \epsilon_4) = \frac{F}{4} \cdot 4\epsilon_{45^\circ}$$

with a *full bridge factor* equal to 4 !



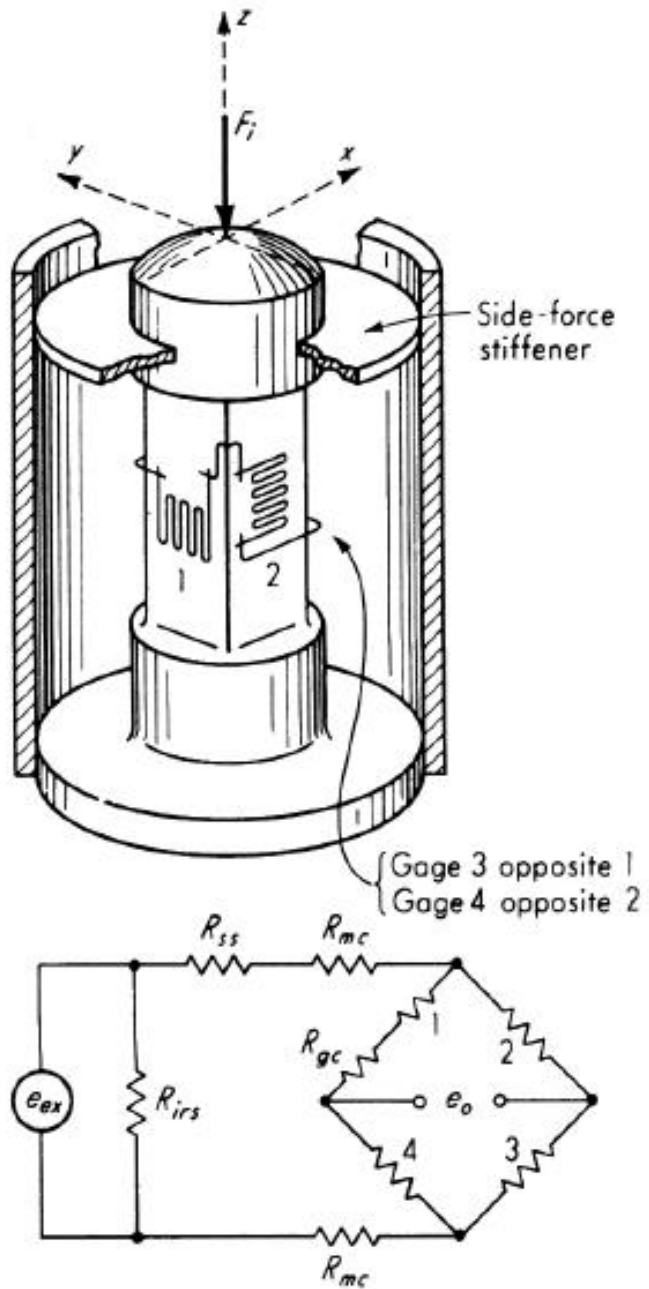
Examples of complete strain gage measurement chains :



Note that strain gage channels need to be *calibrated* before use, this is done by connecting internally shunt resistors R_s that simulate specific strain values:

$$\frac{\Delta R}{R_g} = F \cdot \varepsilon_{elettrica} \quad \text{with} \quad \Delta R = R_g - \frac{R_g R_s}{R_g + R_s} \quad \text{and} \quad \frac{\Delta e}{E} = \frac{1}{4} \frac{\Delta R}{R_g} = \frac{1}{4} (F \cdot \varepsilon_{elettrica})$$

Example of strain gage application in a Load Cell to measure a force



Physical basic equation: $F = k \cdot x$

where (for tension and compression) $k = \frac{AE}{l}$

Graduation curve is linear: $x = \frac{1}{AE} \cdot F$

And sensitivity is constant: $S = \frac{\Delta x}{\Delta F} = \frac{1}{AE}$

